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## Hybrid Force/Motion Control and Internal Dynamics of Quadrotors for Tool Operation

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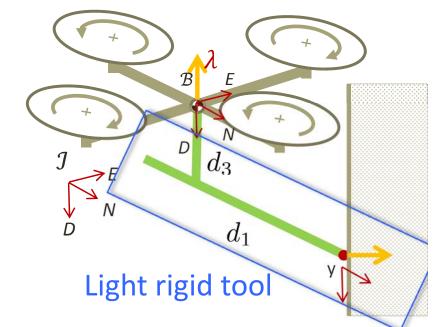
Supported in part by the National Research Foundation (NRF), Korea (2012-R1A2A2A0-1015797)



### Content

- Motivations
- Literature review
- Problem formulation
- Internal dynamics
- Simulation
- Conclusions and future work

### Motivations



- Quadrotor with tool operation
  - Certain tasks require UAV-environment interaction
  - Payload limitation of UAV: prefer light rigid tool to multi-DOF arm

#### Objective

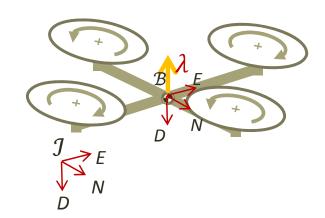
- Control the tool-tip y performing tasks (force/motion control)
- Quadrotor as a versatile robotic platform



### Quadrotor control

#### Motion control

- Trajectory tracking control as M. Hua
   2009
- Cooperative control as *N. Michael* 2011
- Distributed control as D. Lee 2012
- ...



#### Only few works on UAV-environment interaction

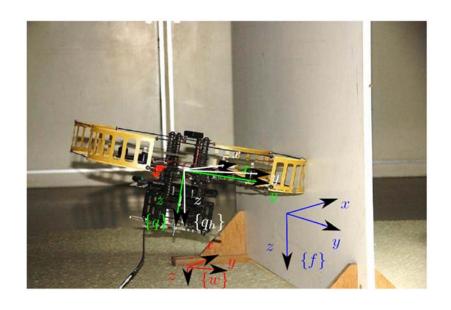
M.-D. Hua, T. Hamel, P. Morin, and C. Samson, "A Control Approach for Thrust-Propelled Underactuated Vehicles and its Application to VTOL Drones," *IEEE Transactions on Automatic Control*, vol. 54, pp. 1837-1853, 2009.

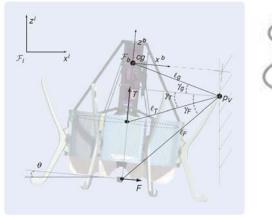
N. Michael, J. Fink, and V. Kumar, "Cooperative manipulation and transportation with aerial robots," Autonomous Robots, vol. 30, pp. 73-86, 2011.

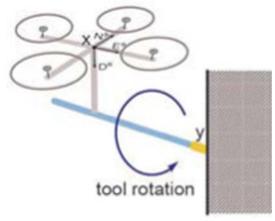
D. Lee, "Distributed backstepping control of multiple thrust-propelled vehicles on a balanced graph," Automatica, vol. 48, pp. 2971-2977, 2012.



### UAV tool operation







# Quasi-static wrench generator Dynamics effect was not taken into account

S. Bellens, J. De Schutter, and H. Bruyninckx, "A hybrid pose / wrench control framework for quadrotor helicopters," in IEEE International Conference on Robotics and Automation (ICRA), 2012, pp. 2269-2274.

#### Planar Dynamics

L. Marconi and R. Naldi, "Control of Aerial Robots: Hybrid Force and Position Feedback for a Ducted Fan," *IEEE Control Systems Magazine*, vol. 32, pp. 43-65, 2012.

D. Lee and C. Ha, "Mechanics and control of quadrotor for tool operation," in ASME Dynamic Systems and Control Conference, 2012, pp. 177-184.

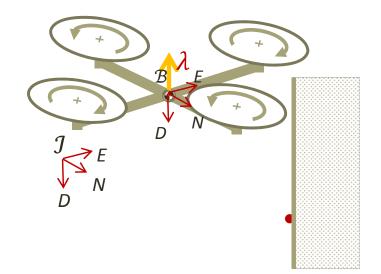




### Problem formulation

#### Model

$$m\ddot{x} = -\lambda Re_3 + mge_3 + f_e$$
$$J\dot{\omega} + \omega \times J\omega = \tau + \tau_e, \dot{R} = RS(\omega)$$





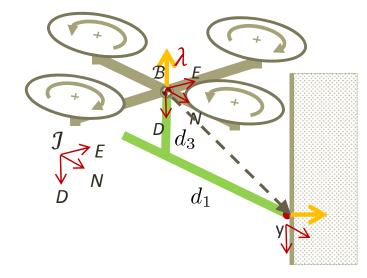
### Problem formulation

Model center-of-mass 
$$m\ddot{x}=-\lambda Re_3+mge_3+f_e$$
 
$$J\dot{\omega}+\omega\times J\omega=\tau+\tau_e, \dot{R}=RS(\omega)$$

#### Tool-tip position

$$y = x + Rd$$

Objective – Hybrid force/motion control



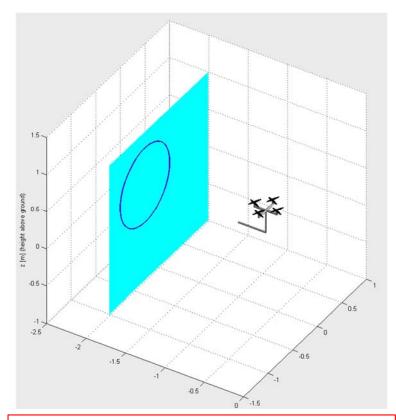
y: tool-tip position in fixed frame  $\mathcal{I}$  d: tool-tip position in body frame  $\mathcal{B}$ 

#### Challenges

Underactuated, yet, need to control center-of-mass position and rotation simultaneous to generate desired motion/force of y



### Contributions



Hybrid force/motion control of quadrotor for tool operation

#### Contributions

- Reveal internal dynamics and elucidate structural condition necessary to internal dynamics stability
- Propose controller to prevent finitetime escape
- Develop hybrid force/motion control

Generalize the result of [Lee&Ha, DSCC12] for planar SE(2) motion to general SE(3) motion





### Problem formulation

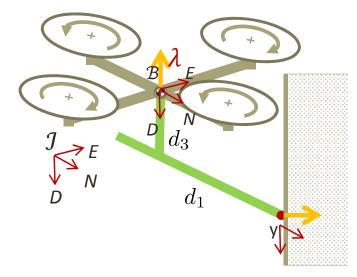
$$m\ddot{x} = -\lambda Re_3 + mge_3 + f_e$$

$$J\dot{\omega} + \omega \times J\omega = \tau + \tau_c, \dot{R} = RS(\omega)$$

#### Tool-tip position

$$y = x + Rd$$

$$\ddot{x} = \ddot{y} - R \left[ S(\dot{w}) + S^2(w) \right] d$$



y: tool-tip position in fixed frame  ${\mathcal I}$ 

d: tool-tip position in body frame  ${\mathcal B}$ 

y-control needs simultaneous control of x and rotation R

### y-dynamics

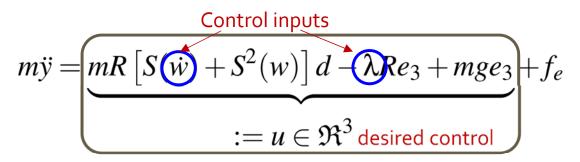
$$m\ddot{y} = mR \left[ S(\dot{w}) + S^2(w) \right] d - \lambda Re_3 + mge_3 + f_e$$

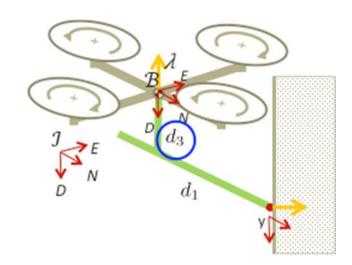
$$J\dot{w} + w \times Jw = \tau + \tau_c, \quad \dot{R} = RS(w)$$
control torque



### Problem formulation

### y-dynamics





<u>Prop. 1</u> [DSCC12] Any control command u can be generated as long as  $d_3 \neq 0$  (e.g., if d3=0, quadrotor cannot drive y-point forward instantaneously)

*Internal dynamics* Define dynamic relationship btw  $\dot{\omega}, \omega$  and R

$$S(d)\dot{\omega} + S(\omega)S(d)\omega + \frac{\lambda}{m}e_3 - gR^Te_3 = -\hat{u}_d$$
 Relation in so(3) Relation in SO(3)

$$\hat{u}_d = \frac{1}{m} R^T u$$





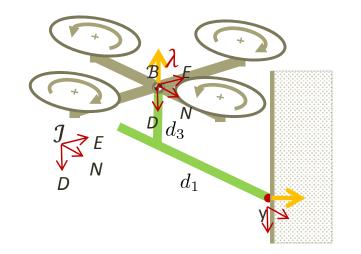
### Internal dynamics

u given

$$S(d)\dot{\omega} + S(\omega)S(d)\omega + \frac{\lambda}{m}e_3 - gR^Te_3 = -\hat{u}_d$$

S(d) is singular quadratic term SO(3)

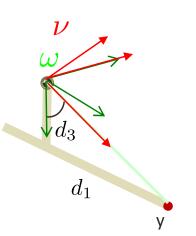
$$S(d) = \begin{bmatrix} 0 & -d_3 & 0 \\ d_3 & 0 & -d_1 \\ 0 & d_1 & 0 \end{bmatrix}$$



Using a coordinate transformation

$$\omega = [\Sigma_{\top} \quad \Sigma_{\perp}] \nu = \Sigma \nu \qquad \Sigma \coloneqq \frac{1}{\alpha} \begin{bmatrix} -d_3 & 0 & d_1 \\ 0 & \alpha & 0 \\ d_1 & 0 & d_3 \end{bmatrix}$$

$$\begin{bmatrix} -d_3\dot{\nu}_2 \\ -\alpha\dot{\nu}_1 \\ d_1\dot{\nu}_2 \end{bmatrix} + \begin{bmatrix} d_1\nu_1^2 + d_1\nu_2^2 + d_3\nu_1\nu_3 \\ -\alpha\nu_2\nu_3 \\ d_3\nu_1^2 + d_3\nu_2^2 + d_1\nu_1\nu_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{\lambda}{m} \end{bmatrix} - gR^Te_3 = -\hat{u}_d$$





### Internal dynamics

#### Internal dynamics in u

$$\begin{bmatrix}
-d_3\dot{\nu}_2 \\
-\alpha\dot{\nu}_1 \\
d_1\dot{\nu}_2
\end{bmatrix} + \begin{bmatrix}
d_1\nu_1^2 + d_1\nu_2^2 + d_3\nu_1\nu_3 \\
-\alpha\nu_2\nu_3 \\
d_3\nu_1^2 + d_3\nu_2^2 + d_1\nu_1\nu_3
\end{bmatrix} + \begin{bmatrix}
0 \\
0 \\
\frac{\lambda}{m}
\end{bmatrix} - gR^T e_3 = -\hat{u}_d$$
SO(3)

#### Local parameterization

$$\frac{d}{dt}[\phi, \theta, \psi]^T = \Gamma\omega = \Gamma\Sigma\nu$$

$$\frac{d}{dt}[\phi,\theta,\psi]^T = \Gamma\omega = \Gamma\Sigma\nu \qquad \Gamma(\theta,\psi) := \begin{bmatrix} 0 & \sin\psi & \cos\psi \\ 0 & \cos\theta & \cos\theta \\ 0 & \cos\psi & -\sin\psi \\ 1 & \frac{\sin\theta\sin\psi}{\cos\theta} & \frac{\sin\theta\cos\psi}{\cos\theta} \end{bmatrix}$$

#### Full Internal Dynamics

$$\frac{d}{dt}(\theta, \psi, \nu_1, \nu_2)^T = \mathcal{F}(\theta, \psi, \nu_1, \nu_2) + (0, 0, \frac{1}{\alpha}\hat{u}_{d2}, \frac{1}{d_3}\hat{u}_{d1})^T$$

$$\mathcal{F}_1 = -\frac{d_1}{\alpha}\nu_1 \operatorname{s} \psi + \nu_2 \operatorname{c} \psi - \frac{d_3}{\alpha}\nu_3 \operatorname{s} \psi$$

$$\mathcal{F}_2 = -\frac{d_3}{\alpha}\nu_1 + \frac{d_1}{\alpha}\nu_1 \operatorname{t} \theta \operatorname{c} \psi + \nu_2 \operatorname{t} \theta \operatorname{s} \psi + \frac{d_1}{\alpha}\nu_3 + \frac{d_3}{\alpha} \operatorname{t} \theta \operatorname{c} \psi \nu_1 \nu_3$$

$$\mathcal{F}_3 = -\nu_2 \nu_3 - \frac{g}{\alpha} \operatorname{c} \theta \operatorname{s} \psi$$

$$\mathcal{F}_4 = -\gamma(\nu_1^2 + \nu_2^2) + \nu_1 \nu_3 + \frac{g}{d_3} \operatorname{s} \theta$$





### Linearized internal dynamics

$$\frac{d}{dt}(\theta, \psi, \nu_1, \nu_2)^T = \mathcal{F}(\theta, \psi, \nu_1, \nu_2) + (0, 0, \frac{1}{\alpha}\hat{u}_{d2}, \frac{1}{d_3}\hat{u}_{d1})^T$$

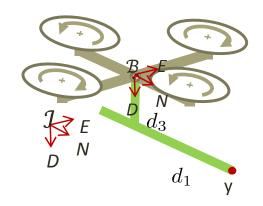
#### Equilibrium and linearization

At 
$$[\theta, \psi, \nu_1, \nu_2] = [0, 0, 0, 0]$$

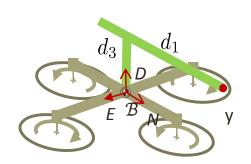
$$\begin{bmatrix} \dot{\theta} \\ \dot{\psi} \\ \dot{\nu}_1 \\ \dot{\nu}_2 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 & 1 \\ 0 & 0 & -\frac{d_3}{\alpha} & 0 \\ 0 & -\frac{g}{\alpha} & 0 & 0 \\ \frac{g}{d_3} & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} \theta \\ \psi \\ \nu_1 \\ \nu_2 \end{bmatrix} \qquad \lambda_1^2 = \frac{g}{d_3}, \quad \lambda_2^2 = \frac{gd_3}{\alpha^2}$$
 unstable for positive  $d_3$ 

$$\lambda_1^2 = \frac{g}{d_3}, \quad \lambda_2^2 = \frac{g d_3}{\alpha^2}$$

$$d_3 > 0$$



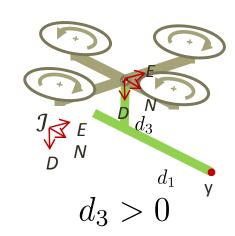
$$\frac{\text{At } [\theta,\psi,\nu_1,\nu_2]=[0,\pi,0,0]}{\lambda_1^2=-\frac{g}{d_3}} \qquad \text{unstable for negative $d_3$}$$





### Linearized internal dynamics

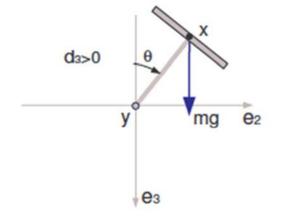
Theorem 1: A necessary condition for the internal dynamics stability at the equilibrium  $(\theta, \psi, \nu_1, \nu_2) = 0$  is  $d_3 < 0$  (i.e., tool attached above the quadrotor)

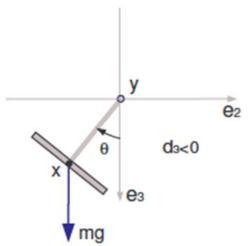


#### Pendulum-like dynamic behavior

At 
$$[\theta, \psi, \nu_1, \nu_2] = [0, 0, 0, 0]$$
 and  $\dot{\nu}_1 = 0$ 

$$\ddot{\theta} = -\gamma \dot{\theta}^2 + \frac{g}{d_3} sin\theta$$
$$\gamma = -\frac{d_1}{d_2}$$







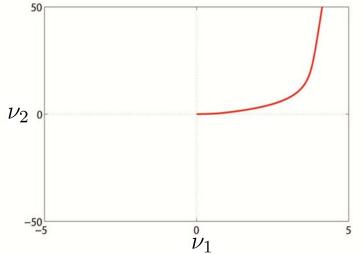
### Finite-time escape

$$\begin{bmatrix} -d_3 \dot{\nu}_2 \\ -\alpha \dot{\nu}_1 \\ d_1 \dot{\nu}_2 \end{bmatrix} + \begin{bmatrix} d_1 \nu_1^2 + d_1 \nu_2^2 + d_3 \nu_1 \nu_3 \\ -\alpha \nu_2 \nu_3 \\ d_3 \nu_1^2 + d_3 \nu_2^2 - d_1 \nu_1 \nu_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{\lambda}{m} \end{bmatrix} - gR^T e_3 = -\hat{u}_d$$

In angular velocity domain

$$\begin{bmatrix} \dot{\nu}_1 \\ \dot{\nu}_2 \end{bmatrix} = \begin{bmatrix} 0 \\ -\gamma(\nu_1^2 + \nu_2^2) \end{bmatrix} + \begin{bmatrix} -\nu_2 \\ \nu_1 \end{bmatrix} \nu_3 + \begin{pmatrix} \bar{u}_1 \\ \bar{u}_2 \end{pmatrix}$$
FINITE TIME ESCAPE 
$$\gamma = -\frac{d_1}{d_2}$$

FINITE TIME ESCAPE 
$$\gamma = -rac{d_1}{d_3}$$



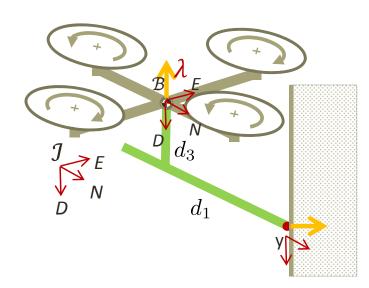
### Finite-time escape

$$\begin{bmatrix} -d_3 \dot{\nu}_2 \\ -\alpha \dot{\nu}_1 \\ d_1 \dot{\nu}_2 \end{bmatrix} + \begin{bmatrix} d_1 \nu_1^2 + d_1 \nu_2^2 + d_3 \nu_1 \nu_3 \\ -\alpha \nu_2 \nu_3 \\ d_3 \nu_1^2 + d_3 \nu_2^2 - d_1 \nu_1 \nu_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{\lambda}{m} \end{bmatrix} - gR^T e_3 = -\hat{u}_d$$

In angular velocity domain

$$\begin{bmatrix} \dot{\nu}_1 \\ \dot{\nu}_2 \end{bmatrix} = \begin{bmatrix} 0 \\ -\gamma(\nu_1^2 + \nu_2^2) \end{bmatrix} + \begin{bmatrix} -\nu_2 \\ \nu_1 \end{bmatrix} \nu_3 + \begin{pmatrix} \bar{u}_1 \\ \bar{u}_2 \end{pmatrix}$$
 Finite time escape 
$$\gamma = -\frac{d_1}{d_3}$$

- Two mechanisms of internal instability:
  - 1) unstable if  $d_3 > o$  (tool below)
  - 2) finite-time escape
- Finite-escape more probable if tool is designed s.t.,  $d_1$  is longer than  $d_3$





### Finite-time escape

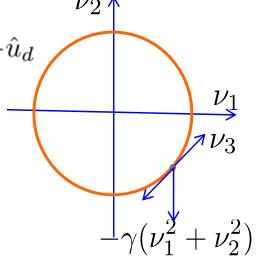
$$\begin{bmatrix} -d_3\dot{\nu}_2 \\ -\alpha\dot{\nu}_1 \\ \frac{1}{d_1\dot{\nu}_2} \end{bmatrix} + \begin{bmatrix} d_1\nu_1^2 + d_1\nu_2^2 + d_3\nu_1\nu_3 \\ -\alpha\nu_2\nu_3 \\ \frac{1}{d_3\nu_1^2 + d_3\nu_2^2 - d_1\nu_1\nu_3} \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \frac{\lambda}{m} \end{bmatrix} - gR^Te_3 = -\hat{u}_d$$

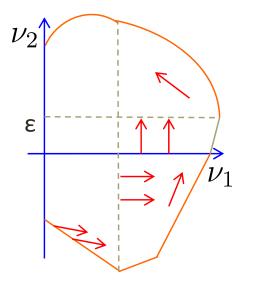
In angular velocity domain

$$\begin{bmatrix} \dot{\nu}_1 \\ \dot{\nu}_2 \end{bmatrix} = \begin{bmatrix} 0 \\ -\gamma(\nu_1^2 + \nu_2^2) \end{bmatrix} + \begin{bmatrix} -\nu_2 \\ \nu_1 \end{bmatrix} \nu_3 + \begin{pmatrix} \bar{u}_1 \\ \bar{u}_2 \end{pmatrix}$$
 Solve the property of the proof of the

Propose: Finite-time escape preventing action (in the sense of boundedness of  $\nu$ )

$$\nu_3 = k\nu_1(1+\nu_2^2) \quad k \ge \gamma$$







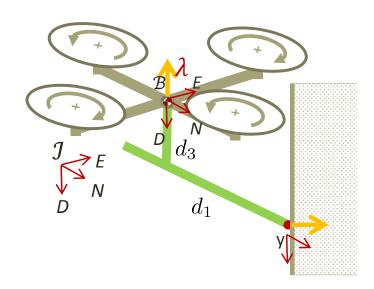
### Hybrid force/motion control

#### Model

$$m\ddot{y} = u + f_e$$

$$h(y) = 0 \quad \frac{\partial h}{\partial y}\dot{y} = 0$$

Hybrid force/motion control require the contact to be maintained all the time, which is not practical





#### Passive decomposition [Lee&Li, IEEE T-AC2013]

$$\dot{y} = \Delta v = \begin{bmatrix} \Delta_{\top} & \Delta_{\bot} \end{bmatrix} \begin{pmatrix} v_l \\ v_h \end{pmatrix} \quad u + f_e = \begin{bmatrix} \Omega_{\top}^T & \Omega_{\bot}^T \end{bmatrix} \begin{pmatrix} u_l + f_l \\ u_h + f_h \end{pmatrix}$$

$$m\dot{v}_l + Q_l v_l + Q_{lh} v_h = u_l + f_l$$

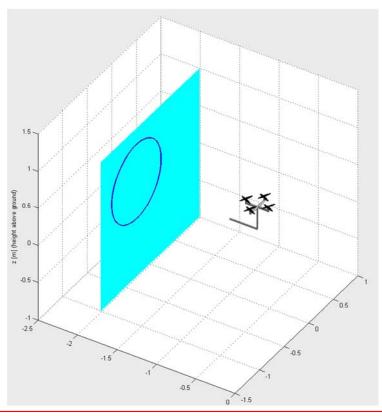
$$m\dot{v}_h + Q_h v_h + Q_{hl} v_l = u_h + f_h$$

The system is decomposed into tangential motion subspace and normal force subspaces and then suitably controlled.

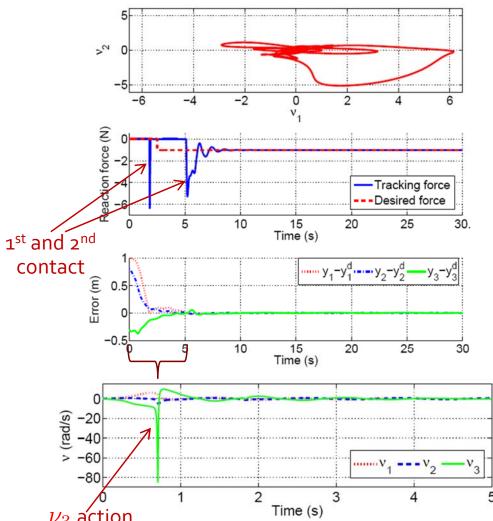




### Simulation



Upside down effect d = [0.35; 0; 0.3]

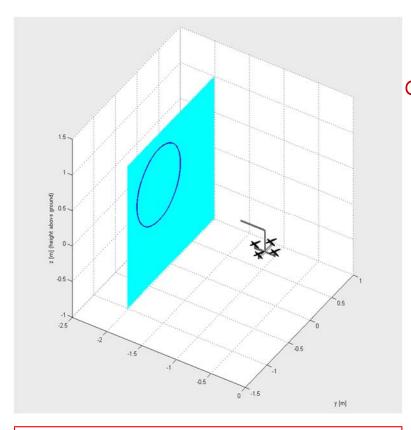


Upside down effect

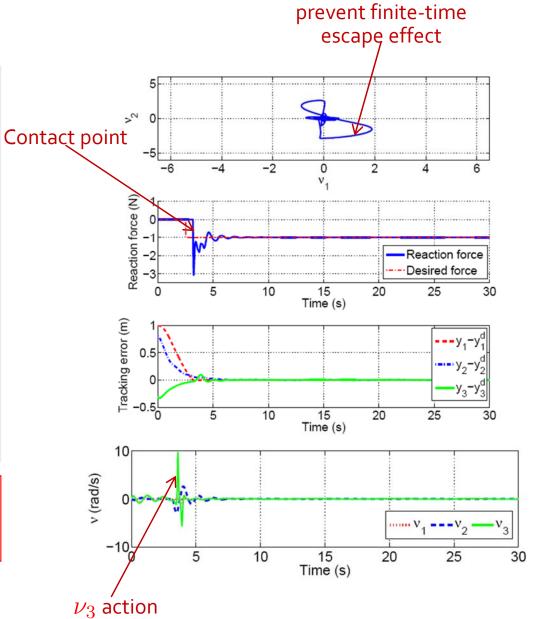




### Simulation

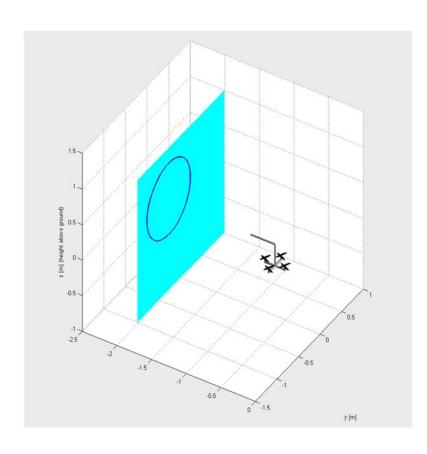


Hybrid force/motion control of quadrotor for tool operation d = [0.35; 0; -0.3]:





### Conclusions and future work



#### Physical interaction

- Reveal internal dynamics with necessary condition
- Propose a control action to prevent finite-time escape
- Use passive decomposition for hybrid force/motion control

#### Future work

- Collision-avoidance
- Consider disturbance
- Experimental implementation



# Thank you for your attention!

